

IN THE SPECIFICATION

Please amend the specification by replacing paragraphs [0035], [0036], [0046], [0047], [0049], [0058], [0060], [0072], [0082], [0083], [0097], [0110] and [0119] with replacement paragraphs as follows:

[0035] $Fa = 4*a*t/S^2$ $4*\alpha*t/S^2$, where

[0036] ~~$\alpha = \lambda/(p*c)$~~ $\alpha = \lambda/(\rho*c)$ = the thermal diffusivity constant

[0038] ~~p = density~~ ρ = density

[0046] $\Delta T = \Delta T_o (1 + 0.15*Fo^{-1.9})$ $(1 + 0.15*Fo^{-1.9})$ (1)

[0047] $\Delta T_o = 2*E/(p*c*S)$ $2*E/(\rho*c*S)$, where

[0049] ~~p = density~~ ρ = density

[0058] where $t/t_o = Fo$, t being = the duration of the thermal pulse, and $t_o = S^2/(4*a)$ $S^2/(4*\alpha)$ being a constant characteristic of

[0060] $\Delta T/\Delta T_o - 1 = 1/A$ (4)

[0072] From an initial temperature T_u (Figure 4), we have a surface temperature ~~$T_{u+\Delta T}$~~ $T_{u+\Delta T}$ on the brake disk immediately after braking. However, the temperature in the disk is evened out quickly to $T_{u+\Delta T_o}$, which represents the temperature at which the cooling process begins. The temperature difference between the brake disk and its environment when cooling begins is therefore $T_{u+\Delta T_o} - T_k$, where T_k is the temperature of the cooling element. If the time until the next braking is t_n , we have the temperature T_n when the next braking begins.

[0082] Calculation of Remaining/Consumed Life: Figure 5 shows a relationship between maximum total temperature and number of braking cycles for wearing-out using log-log scales. The relationship consists of two linear functions [0] O, P with different slopes. The reason why two curves are used is that the lining on the brake disk is broken down at high temperature and has a tendency to char. This is because, at high temperatures for linings made of paper, a chemical process, carbonization, takes place. The upper curve [0] O, on the left in the figure, describes the strength in a brake disk, the lining of which has reached such a high temperature that charring has started.

[0083] The slope of the curves and the break-point between the upper curve [0] O and the lower curve Pare obtained from rig testing. The slope of the left, upper curve [0] O may, however, be difficult to produce with great accuracy and, in such a case, it can be estimated with, for example, the Arrhenius function.

[0097] D is damage value per unit of time or distance (damage per hour or damage per kilometer), and n_1 and n_2 are the number of braking cycles per temperature level and unit of time or distance.

[0110] $St = \frac{dT \cdot a \cdot E}{(1-v)} \frac{dT \cdot \alpha \cdot E}{(1-v)}$

[0119] The following background documents are hereby expressly incorporated for purposes of disclosure in the present application, and for reference by concerned persons skilled in the relevant art:

[1] Lauster, E. and Staberoh, U. "Wametechnische Berechnungen bei Lamellenkupplungen" VDI-Z 115 (1973);

[2] Kruger, H. "Das Temperaturverhalten der nassen Lamellenkupplungen" Konstruktion 17 (1963);

[3] Tataiah, K. "An Analysis of Automatic Transmission Clutch-Plate Temperatures" SAE 720287;

[4] Roark, RaymondJ. "Formulas for stress and strain."